Grating Mount Requirements

- Weight: <75 Lbs including grating (43.1 Lbs)
- Size: Fit within structural envelope. 1.5" for cell behind grating.
- Alignment: Adjust in x,y,z tilts and clocking then fix.
- Deflections during an observation:
 - Piston: <10 microns
 - Tilts < 5 🕅-rad
- Surface Figure
 - Grating is 1/8 ms surface = 41nm-rms
 - I assume we must not degrade this by more than 10% so the mounting distortion must not exceed 18 nm-rms surface.
- Active Tilt about short axis. De-coupled from piston. Set and forget.
 - Range > 2.5 m-rad
 - Resolution better than 0.2 m-rad

Grating Mount Requirements, Cont.

- Handling and Installation:
 - Shipping clamps if required for 5 g load.
 - Set down pads in cell.
 - Lifting provisions (threaded fittings for eyebolts).
- Safety Features:
 - Cover
 - Safety Clips
 - Break-aways on flex rods.
- Metrology Provisions:
 - At least 6 gage points must be provided to locate the grating relative to the cell.

Overall View of Cell





3

Cell Details





Tilt Drive Mechanism



Gage Port Locations



Axial Gage Port Locations



8

Cell Envelope Problems



Cell Envelope Violations



Cell Envelope Violations



Cell Envelope Violations



Cell Relieved for Clearance



Cell Relieved for Clearance



Cell Relieved for Clearance



Modified Cell,



16

Second Model of Grating Cell

- Uses solid elements for cores
- Cell mounting flexures modeled with displacement constraints.
- Weight of model (incl 20%) is 35.6 Lbs not including drive (10 lbs at 120%) or grating (41.6 Lbs).



Cell Deflection Results

Requirements are:

In a one hour observation:

Piston < 10 microns

Tilts about X and Y < 5 \square -rad

Cell Deflection Results, 0.25 g					
	Piston	Rx	Ry		
	microns	u-rad	u-rad		
0.25 g Z	2.5	0.17	0.17		
0.25 g X	0.28	3.1	3.4		
0.25 g Y	0.1	2	2.6		
2.5 m-rad	0	0	2538		
Max Deflections including grating mounting:					
	2.8	3.8	4.3		

Cell Mounting, Flexure

- EDM'd flexure provides high stiffness in 5 directions.
 - Flexures are 0.040" thick
 - Force required for 2.5 m-rad rotation is 14 lbs (at 9.5").



Flexure is 1.3" x 3.0"

Cell Mass Properties

- Cell Structure (incl 20% contingency) = 36 Lbs
- Drive mechanism (incl 20 % contingency) 10 Lbs Total 46 Lbs

Acceleration Vector Change in One Hour

- Elevation axis rotation directly effects the instrument.
- Azimuth rotation has no effect on the instrument.
- Derotator motion rotates the instrument relative to the lateral gravity component.
 - Derotator rates are similar to azimuth rates.
 - High derotator rates occur near zenith pointing when the lateral gravity component is small.
- Gravity components have been calculated using a spreadsheet. Results are summarized on the following slides.
 - Lateral g's change by no more than 0.25 g's in an hour (peak rate is about 0.63 g's/hr for a small fraction of an hour).
 - G's along the LOS change no more than 0.22 g's in an hour.

Maximum Gravity Vector Change in One Hour Results over 24 hours for one particular target star.



Observatory Latitude=32°, Target Star is at 25°

Gravity Vector Components, Change in One Hour

Observatory Latitude=32°, Target Star is at 25°

Optical axis is acceleration along the line of sight (LOS).

Lateral is any direction orthogonal to the LOS.



Maximum Gravity Vector Changes in One Hour, Different Star Locations (The Observatory is at 32 deg Latitude)

Note that high derotator rates coincide with high azimuth rates as the LOS passes near zenith but at this time the lateral gravity vector is small. The bump in lateral g's at 32 degrees may be the result of averaging maximum g's.



Grating Mounting

• Requirements:

< 18 nm-rms surface distortion small tilts (~ 1 🕅-rad) small piston (~ 1 🕅-m)

• Achieved:

- 7.1 nm-rms surface
- $0.7 \text{ }\text{m}\text{-rad tilt/g} \quad \dots \quad 0.35 \text{ }\text{m}\text{-rad over } 30 \text{ }\text{$
- 2.6 \mathbb{M} -m piston/g 1.3 \mathbb{M} -m over 30 \mathbb{M}

MAESTRO Grating, Best Location for Axial Supports



MAESTRO Grating, Sub-optimal Axial Support Location



MAESTRO Grating, Sub-optimal Axial Support Location

1 g Sag, 9.3 nm-rms surface



Grating Distortion, Unit Load Cases, Nm-rms Surface

- Analysis results.
 - Piston and tilts removed.
 - Power left in.
 - X=4.824 indicates the axial supports are 4.824" from the C/L.
 - Tilts.
 - 0.41 🕅 -rad for 1 g Z.
 - <0.07 🕅 -rad for X&Y.
 - Axial flex rods will add 0.5 X -rad tilt.
 - net tilt = 0.65 \square -rad

	Unit Load Results					
	X=4.824	X=4.135	X=4.48			
1 g Z	9.3	9.85	5.73			
1 g X	1.95	1.95	1.97			
1 g Y	1.91	1.9	1.91			
At Back						
Fx =1	1.18	1.12	1.16			
Fy=1	0.28	0.27	0.28			
Mx=1	0.14	0.14	0.14			
My=1	0.55	0.53	0.54			
Mz=1	0.02	0.02	0.02			
at Y support						
Fx =1	0.04	0.04	0.04			
Fz=1	4	4.1	4.03			
Mx=1	0.83	0.79	0.81			
My=1	0.55	0.55	0.55			
Mz=1	0.02	0.02	0.02			

Grating Tilt Due To Mounting Compliance

- Using three identical flex rods tilt is ~ 10 [X]-rad (1 g Z)
- Tilt is reduced by:
 - Doubling the stiffness of the 2xForce rod
 - Error forces increase from to (V,M) = (.18,.284) to (V,M) = (.286,.530)
 - Small effect on overall performance (7.06 nm-rms to 7.15 nm-rms)
 - An option is to equalize the load on each axial support
 - Gravity distortion increases from 5.7 to nm-rms to 8.51 nm-rms
 - Tilt (surface fit) increases from 0.41 to 0.53 🕅 -rad

2 - 1 309E+

6 = -8 876F+0

7 = -1.437E+ 8 = -1.986E+



Error Analysis, Gravity and Support Errors, Flex Rod Support

• Results are nm-rms surface

	Unit Load	Est. error	Number	Net Error
	Results	Force	Number	
1 g Z	5.73			5.73
1 g X	1.97			1.97
1 g Y	1.91			1.91
At Back (rod pair)				
Fx =1	1.16	0.361	2	0.59
Fy=1	0.28	0.361	2	0.14
Mx=1	0.14	0.567	2	0.11
My=1	0.54	0.567	2	0.43
Mz=1	0.02	2	2	0.06
At Back (Single rod)				
Fx =1	0.912	0.361	1	0.33
Fy=1	1.49	0.361	1	0.54
Mx=1	0.68	0.567	1	0.39
My=1	0.44	0.567	1	0.25
Mz=1	0.02	2	1	0.04
at Y support				
Fx =1	0.04	0.198	2	0.01
Fz=1	4.03	0.198	2	1.13
Mx=1	0.81	0.946	2	1.08
My=1	0.55	2	2	1.56
Mz=1	0.02	0.946	2	0.03
at X support				
Fx =1	0.05	0.26	1	0.01
Fz=1	3.98	0.26	1	1.03
Mx=1	0.73	2	1	1.46
My=1	0.52	0.986	1	0.51
Mz=1	0.01	0.986	1	0.01
		RSS Net =	7.06	

31

Flex Rod Design and Error Force Estimation

- Axial flexure.
 - The single flexure has double the stiffness (details TBD).
 - Lateral flexures are similar but 6" long with 0.030" thick flexures.



Flex Rod Error Forces

- Use FLEXROD analysis program.
 - Installed alignment errors will not exceed:
 - 0.020" of end to end position error and rod straightness error.
 - 0.5 deg of rod end bend and cell/mirror tilt.
 - Axial force in rods is less than 21.2 lbs except X rod is 42.4 lbs.
 - Estimated error forces are listed on slide 6.

Grating Mount Performance

- Weight:
- Size:

Achieved:

prism.

• Alignment:

Achieved:

< 75 Lbs including grating (43.1 Lbs)Achieved:46+43.1 = 90 Lbs (15 Lbs high).

Fit within structural envelope.1.5" for cell behind grating.*Still need to fix a minor interference with the*

Adjust in x,y,z tilts and clocking then fix. *Adjust x,y and clocking then fix. Fine tune piston and tilts by adjusting grating supports (to cell)* ... ± 0.020 " or ± 4 m-rad range.

- Deflections during an observation:
 - Piston: <10 microns
 - Tilts *Achieved*:

12.5 microns piston 4.3 **▼**-rad tilt

< 5 \mathbb{M} -rad

Grating Mount Performance

- Surface Figure:
 - Grating is 1/8 ms surface = 41nm-rms
 - I assume we must not degrade this by more than 10% so the mounting distortion must not exceed 18 nm-rms surface.

Achieved: 7.1 nm-rms surface

- Active Tilt about short axis. De-coupled from piston. Set and forget.
 - Range > 2.5 m-rad
 - Resolution better than 0.2 m-rad
 - Achieved:
 ~ ±5 m-rad range Resolution ~ 6 ⊠-rad/step Encoding accuracy ~ 0.17 m-rad using a 7 arc-min resolver on the cam shaft. Could achieve 20 ⊠-rad accuracy with a better (more expensive) encoder.

 Note:
 With a simple eccentric cam, cam angle and grating tilt are related through the sine of the cam shaft angle. If a linear function is desired the cam could be cut with a linear rise.

Grating Mount Requirements, Cont.

- Handling and Installation:
 - Shipping clamps if required for 5 g load.
 - Set down pads in cell.
 - Lifting provisions (threaded fittings for eyebolts).
 - *Achieved:*
 - Shipping clamps:
 - 8 built in, replace safety clips with 4 additional restraints load capacity TBD, estimated to be 20 g's
 - Set down pads
 - 3 of the restraints usable for axial set down pads
 - 5 pads in the X-Y directions serve as push pulls for X, Y, Rz
 - Lifting provisions (not shown):
 - Could replace the 4 safety clips with plates threaded for eyebolts or could add ¹/₄ threads to upper surface corners.

Grating Mount Requirements, Cont.

- Safety Features:
 - Cover
 - Safety Clips
 - Break-aways on flex rods.
 - *Achieved:*
 - Cover, Not Shown, would attach to raised flange (possible clearance issue with prism)
 - Safety clips provided.
 - *Tension/compression breakaways on lateral rods, compression only on axial rods.*
- Metrology Provisions:
 - At least 6 gage points must be provided to locate the grating relative to the cell.
 - Achieved: 7

Grating Mount Requirements

- Weight: <75 Lbs including grating (43.1 Lbs)
- Size: Fit within structural envelope. 1.5" for cell behind grating.
- Alignment: Adjust in x,y,z tilts and clocking then fix.
- Deflections during an observation:
 - Piston: <10 microns
 - Tilts < 5 ₩-rad
- Surface Figure
 - Grating is 1/8 ms surface = 41nm-rms
 - I assume we must not degrade this by more than 10% so the mounting distortion must not exceed 18 nm-rms surface.
- Active Tilt about short axis. De-coupled from piston. Set and forget.
 - Range > 2.5 m-rad
 - Resolution better than 0.2 m-rad